



Fig. 6.3.2b—Lumped and distributed soil impedance.

damping values by specifying ground modifications and by changing the proportioning of foundation and structural members/elements. The use of vibration isolation devices provides the ability to establish more precise and stable impedance characteristics, and can capture the benefit of effectively reducing the foundation vibration level.

Vibration isolators are proprietary devices consisting of springs, dampers, or both. Springs are used to reduce the stiffness otherwise provided by the ground/structure. Dampers are used to increase the energy dissipation in the machine-foundation system. These devices can be placed between an inertia block and supporting structural members/elements, as indicated in Fig. 6.3.3a. They can also be placed between a tabletop slab and columns as indicated in Fig. 3.3.4 and 3.3.5.

Isolation systems are used for two purposes. The first purpose is to reduce the transmission of vibrations between a supported machine and the surrounding environment. Isolators could be used with dynamic machinery, such as forging hammers, to prevent machine-induced vibrations from interfering with nearby machines and with worker comfort. The second purpose is to create a less expensive machine support when the cost of installing isolators is less than the cost of the larger structural members/elements that would be required without isolators.

There are three basic spring isolator types used for dynamic machinery applications: rubber (or elastomeric materials) pads, steel springs, and air mounts. Rubber pads are generally the stiffest designs and have a nonlinear load-deflection relationship. Steel springs are less stiff, providing better isolation but low damping. Air-mount types are the least-stiff type of isolator.

Most rubbers and steel springs do not have high level of damping. These designs can be augmented with viscous dampers, providing higher damping levels up to 40 percent, depending on the specific application needs.

The effectiveness of vibration isolators is expressed in two factors: the transmissibility factor and degree of isolation.

The transmissibility factor T_F is the peak force transmitted to the foundation, F_D , divided by the machine's peak dynamic force F . For rotating and reciprocating machines



Fig. 6.3.3a—Isolator-supported fan foundation (photo courtesy of GERB Vibration Control Systems, Inc.).

producing a harmonic dynamic force, the transmissibility factor derived from the equation of motion is indicated in Fig. 6.3.3b for various values of damping ratio D and tuning ratio η . The tuning ratio is defined as the forcing frequency ω_s , divided by the undamped natural frequency ω_n of the machine-inertia block-isolator system. A reduction in the transmissibility occurs for values of η larger than the square root of 2. Stiffness isolation devices typically target η values between 2 to 5. Damping devices are most effective when η is near 1 at resonant condition. Note that in the isolation region for $\eta > \sqrt{2}$, higher damping will result in less isolation. Thus, special care should be taken to select suitable damper devices for optimal design between vibration isolation and vibration mitigation near resonance.

The equation for the transmissibility factor, T_F is expressed as

$$T_F = \left| \frac{F_D}{F} \right| = \left| \sqrt{\frac{1 + (2D\eta)^2}{(1 - \eta^2)^2 + (2D\eta)^2}} \right| \quad (6.3.3a)$$

The isolation efficiency as a percentage, I , is derived from the transmissibility factor as

$$I = 100(1 - T_F) \quad (6.3.3b)$$

Foundations For Dynamic Equipment Inti

Michael Brown



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